

MEMBER REPORT FOR LIPPED CHANNEL SECTION

Design Provision used: AS/NZS 4600:2005

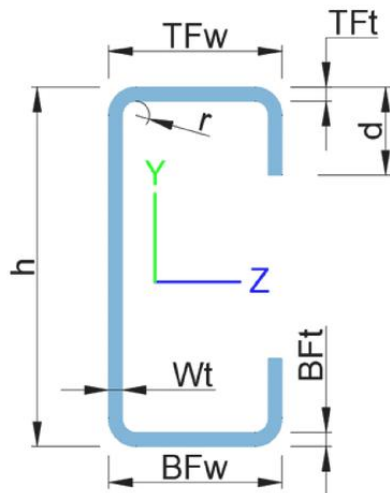
PROJECT DETAILS

Project Name:
Project ID:
Company:
Designer:
Client:
Project Notes:
Project Units: Metric

General member design information

Section Name: C10010

Shape: C-Section w/ Lips



Dimensions:

Height, $h = 102 \text{ mm}$
 Web Thickness, $Wt = 1 \text{ mm}$
 Top Flange Width, $TFw = 51 \text{ mm}$
 Top Thickness, $TFt = 1 \text{ mm}$
 Bottom Flange Width, $BFw = 51 \text{ mm}$
 Bottom Thickness, $BFt = 1 \text{ mm}$
 Lip Depth, $d = 12.5 \text{ mm}$
 Fillet Radius, $r = 5 \text{ mm}$

Properties:

Area, $A = 216 \text{ mm}^2$
 Moment of Inertia about the z-axis, $Iz = 364000 \text{ mm}^4$
 Moment of Inertia about the Y-Axis, $Iy = 75500 \text{ mm}^4$
 Plastic Section Modulus about the z-axis, $Zz = 9814.276 \text{ mm}^3$
 Plastic Section Modulus about the Y-Axis, $Zy = 4181.586 \text{ mm}^3$
 Torsion Constant, $J = 71.9 \text{ mm}^4$
 Warping Constant, $Iw = 160000000 \text{ mm}^6$

Material properties:

Material Name: **Australian/New Zealand Standard (AS/NZS) - AS/NZS 1594 - HA300, HU300 - Standard**
 Modulus of Elasticity, $E = 200000 \text{ mpa}$
 Yield Strength, $Fy = 300 \text{ mpa}$
 Ultimate Tensile Strength, $Fu = 400 \text{ mpa}$

Flexural Buckling parameters:

Member length for flexural buckling, $L = 3625 \text{ mm}$
 Length between braced points, $Lb = 3625 \text{ mm}$

Lateral Torsional Buckling parameters:

Coefficient for lateral-torsional buckling, $Cs = -1$
 End moment coefficient in interaction formula, $CTf = 1.0$
 Bending coefficient dependent on moment gradient, $Cb = 1.0$
 Member length for Lateral Torsional Buckling, $L = 3625 \text{ mm}$

Design Internal Forces

Load Case:

Name = **Worst Case Load Combination**

For check axial strength:

Absolute Maximum Axial Force, **P** = 56.4 N

For check flexural strength about Z-Axis:

Absolute Maximum Major Bending Moment, **Mz** = 40 N-mm

For check flexural strength about Y-Axis:

Absolute Maximum Major Bending Moment, **My** = 19470 N-mm

For check shear strength Y-Axis:

Absolute Maximum Shear Force, **Vx** = 0.01 N

For check shear strength Z-Axis:

Absolute Maximum Shear Force, **Vy** = 5.37 N

For check interaction of combined compression and bending strength:

Axial Force, **P** = 56.4 N

Z-Axis Bending Moment, **Mz** = 40 N-mm

Y-Axis Bending Moment, **My** = 19470 N-mm

For check interaction of combined bending and shear strength:

Z-Axis Bending Moment, **Mz** = 40 N-mm

Y-Axis Bending Moment, **My** = 19470 N-mm

Shear Force, **Vx** = 5.37 N

Shear Force, **Vy** = 0.01 N

BENDING CAPACITY

Bending about Y-Axis

3.3.2 Nominal section moment capacity, Ms (Section 3.3.2) Clause 3.3.2.2

$$M_{sy} = Z_{yc} F_y$$

$$M_{sy} = (2033.16) (300) = 609948 \text{ N} - \text{mm}$$

Calculate Nominal flexural strength about Y-Axis (M_{sy})

$$\text{Design Flexural Strength} = M_{sy} \phi_b = (609948) (0.9) = 548953.2 \text{ N} - \text{mm}$$

$$\frac{M_y^*}{\text{Design Flexural Strength}} = \frac{19470}{548953.2} = 0.035 \leq 1.0$$

Bending about Z-Axis

3.3.2 Nominal section moment capacity, Ms (Section 3.3.2) Clause 3.3.2.2

$$M_{sz} = Z_{zc} F_y$$

$$M_{sz} = (5811.98) (300) = 1832339 \text{ N} - \text{mm}$$

Calculate Nominal flexural strength about Z-Axis (M_{sz})

$$\text{Design Flexural Strength} = M_{sz} \phi_b = (1832339) (0.9) = 1649105.1 \text{ N} - \text{mm}$$

$$\frac{M_z^*}{\text{Design Flexural Strength}} = \frac{40}{1649105.1} = 0 \leq 1.0$$

FLEXURAL BUCKLING CAPACITY

Concentrically loaded compression members, fox (Section 3.4) Clause 3.3.3.2(14)

$$\frac{L_x}{r_x} = \frac{3625}{41.051} = 88.305$$

$$\frac{L_y}{r_y} = \frac{3625}{18.696} = 193.892$$

Since $\frac{L_y}{r_y} > \frac{L_x}{r_x}$, Euler buckling about the y-axis will control.

Determine controlling euler buckling

$$f_{oc} = \frac{\pi^2 E}{(l_y/r_y)^2} = \frac{\pi^2 (200000)}{[(3625) / (18.696)]^2} = 52.506 \text{ mpa}$$

Calculate Elastic Buckling Stress (f_{oc})

$$\lambda_c = \sqrt{\frac{F_y}{f_{oc}}} = \sqrt{\frac{300}{52.506}} = 2.39$$

Calculate Slenderness factor (λ_c)

$$f_n = (0.877/(\lambda_c)^2) F_y = (0.877/(2.39)^2) 300 = 46.048 \text{ mpa}$$

Calculate Critical Stress (f_n)

$$N_c = A_e f_n = (216)(46.048) = 9946.27 \text{ N}$$

Calculate Nominal axial strength (N_c)

$$\text{Design Strength} = N_c \phi = (9946.27)(0.85) = 8454.329 \text{ N}$$

$$\frac{N^*}{\text{Design Strength}} = \frac{56.4}{8454.329} = 0.007 \leq 1.0$$

COMBINED AXIAL COMPRESSION AND BENDING CHECK

Combined axial compression and bending, (Section 3.5.1) Clause 3.5.1

$$\frac{N^*}{\phi_c N_s} + \frac{C_{mx} M_x^*}{\phi_b M_{sx} \alpha_x} + \frac{C_{my} M_y^*}{\phi_b M_{sy} \alpha_y} = \frac{56.4}{0.85 (29748.108)} + \frac{1 (40)}{0.9 (1832339) (0.999)} + \frac{1 (19470)}{0.9 (609948) (0.994)} =$$

$$\frac{N^*}{\phi_c N_c} + \frac{M_x^*}{\phi_b M_{sx}} + \frac{M_y^*}{\phi_b M_{sy}} = \frac{56.4}{0.85 (41648.4)} + \frac{40}{0.9 (1832339)} + \frac{19470}{0.9 (609948)} = 0.037 \leq 1.0$$

FLEXURAL TORSIONAL BUCKLING CAPACITY CHECK

3.4 Concentrically loaded compression members SECTION 3.4.3

$$f_{ozz} = \frac{1}{2\beta} \left((f_{ox} + f_{oz}) - \sqrt{(f_{ox} + f_{oz})^2 - 4\beta f_{ox} f_{oz}} \right) = \frac{1}{2(0.558)} \left((253.14 + 37.803) - \sqrt{(253.14 + 37.803)^2 - 4(0.558)(253)} \right)$$

$$\beta = 1 - (x_o/r_{o1})^2 = 1 - (-40.164/60.398)^2 = 0.558$$

$$f_{ox} = \frac{\pi^2 E}{(l_{ex}/r_x)^2} = \frac{\pi^2 (200000)}{(3625/41.051)^2} = 253.14 \text{ mpa}$$

$$f_{oz} = \frac{GJ}{Ar_{o1}^2} \left(1 + \frac{\pi^2 EI_w}{GJl_{ez}^2} \right) = \frac{(80000)(71.9)}{(216)(60.398)^2} \left(1 + \frac{\pi^2 (200000)(160000000)}{(80000)(71.9)(3625)^2} \right) = 37.803 \text{ mpa}$$

Calculate Flexural-Torsional Buckling (f_{ozz})

$$x = \sqrt{\frac{2}{3}} = \sqrt{\frac{2 \cdot 3}{3 \cdot 3}} = \frac{\sqrt{6}}{3}$$

Substituting in:

$$x = \left(\frac{\sqrt{6}}{3}\right)^2 = \left(\frac{6}{9}\right) = \frac{2}{3}$$

Substituting in:

$$x = \frac{2}{3} = \frac{2 \cdot 2}{3 \cdot 2} = \frac{4}{6}$$

Substituting in:

$$\frac{4}{6} = \frac{4 \cdot 2}{6 \cdot 2} = \frac{8}{12}$$

$$\frac{8}{12} = \frac{2 \cdot 4}{3 \cdot 4} = \frac{2}{3}$$

EXERCISES: SOLVING EQUATIONS WITH RADICALS

1. $x = \frac{1}{2}$

$$x = \frac{1}{2} \text{ is the solution of } x = \frac{1}{2} \text{ since } \frac{1}{2} = \frac{1}{2} \text{ is true.}$$

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$$x = \left(\frac{1}{2}\right)^2 = \frac{1}{4}$$

$$x = \left(\frac{1}{4}\right)^2 = \left(\frac{1 \cdot 1}{4 \cdot 4}\right) = \frac{1}{16}$$

$$x = \frac{1}{16} = \frac{1 \cdot 2}{16 \cdot 2} = \frac{2}{32}$$

Substituting in:

$$x = \left(\frac{2}{32}\right) = \frac{2}{32} \text{ since } \frac{2}{32} = \frac{2}{32} \text{ is true.}$$

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Assume $\mathbf{u} = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$

$$\mathbf{u} = \frac{1}{\sqrt{2}} \begin{bmatrix} 1 & 1 \\ 1 & -1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = \frac{1}{\sqrt{2}} \begin{bmatrix} u_1 + u_2 \\ u_1 - u_2 \end{bmatrix}$$

$$u_1 = \frac{1}{\sqrt{2}}(u_1 + u_2)$$

$$u_2 = \frac{1}{\sqrt{2}}(u_1 - u_2)$$

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Assume $\mathbf{u} = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$

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PROBLEM 10: Find the orthogonal matrix \mathbf{Q} and the diagonal matrix \mathbf{D} .

10.

$$\mathbf{A} = \begin{bmatrix} 1 & 1 \\ 1 & -1 \end{bmatrix} \text{ is a } 2 \times 2 \text{ matrix. Find the orthogonal matrix } \mathbf{Q} \text{ and the diagonal matrix } \mathbf{D} \text{ such that } \mathbf{A} = \mathbf{Q}\mathbf{D}\mathbf{Q}^T.$$

Assume $\mathbf{u} = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$

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$$u_1 = \frac{1}{\sqrt{2}}(u_1 + u_2)$$

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$$\mathbf{u} = \frac{1}{\sqrt{2}} \begin{bmatrix} 1 & 1 \\ 1 & -1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = \frac{1}{\sqrt{2}} \begin{bmatrix} u_1 + u_2 \\ u_1 - u_2 \end{bmatrix}$$

Assume $\mathbf{u} = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$

Example 10.1.1: Integration by Substitution

$$\int 2x \sqrt{x^2 + 1} \, dx = \int u^{1/2} \cdot \frac{1}{2} du = \frac{2}{3} u^{3/2} + C = \frac{2}{3} (x^2 + 1)^{3/2} + C$$

$$= \frac{2}{3} (x^2 + 1)^{3/2} + C$$

$$= \frac{2}{3} (x^2 + 1)^{3/2} + C$$

Check: Differentiate the result to see if you get the original integrand.

$$\frac{d}{dx} \left(\frac{2}{3} (x^2 + 1)^{3/2} + C \right) = 2x \sqrt{x^2 + 1}$$

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10.1.1